2006 International Energy Conservation Code Errata

(Portions of text and tables not shown are unaffected by the errata)

1st thru 8th PRINTING (Posted: October 7, 2011)

2006 IECC

CHAPTER 5

COMMERCIAL ENERGY EFFICIENCY
Table 503.2.3(8)

b.Condenser ΔT = Leaving condenser water temperature (°F)-Entering condenser water temperature (°F).
$K_{adj} = 6.1507 – 0.30244(X) + 0.0062692(X^2) – 0.000045595 (X^3)$
Where: X = Condenser ΔT + Lift
$COP_{adj} = K_{adj} \times COP_{std}$

Table 503.2.3(9)

b. Condenser ΔT = Leaving condenser water temperature (°F) – Entering condenser water temperature (°F).
$K_{adj} = 6.1507 – 0.30244(X) + 0.0062692(X)^2 – 0.000045595 (X^3)$
Where: X = Condenser ΔT + Lift $COP_{adj} = K_{adj} \times COP_{std}$

Table 503.2.3(10)

b. Condenser ΔT = Leaving condenser water temperature (°F) – Entering condenser water temperature (°F).
$K_{adj} = 6.1507 – 0.030244(X) + 0.0062692(X)^2 – 0.000045595 (X^3)$
Where: X = Condenser ΔT + Lift
$COP_{adj} = K_{adj} \times COP_{std}$

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503.2.7.1.3 High-pressure duct systems. Ducts designed to operate at static pressures in excess of 3 inches w.g. (746 Pa) shall be insulated and sealed in accordance with Section 503.2.7. In addition, ducts and plenums shall be leak-tested in accordance with the SMACNA *HVAC Air Duct Leakage Test Manual* with the rate of air leakage (CL) less than or equal to 6.0 as determined in accordance with Equation 5-2.

~~$CL = F \times P^{0.65}$~~ $CL = F / P^{0.65}$ (Equation 5-2)

(No change to remainder of section)